



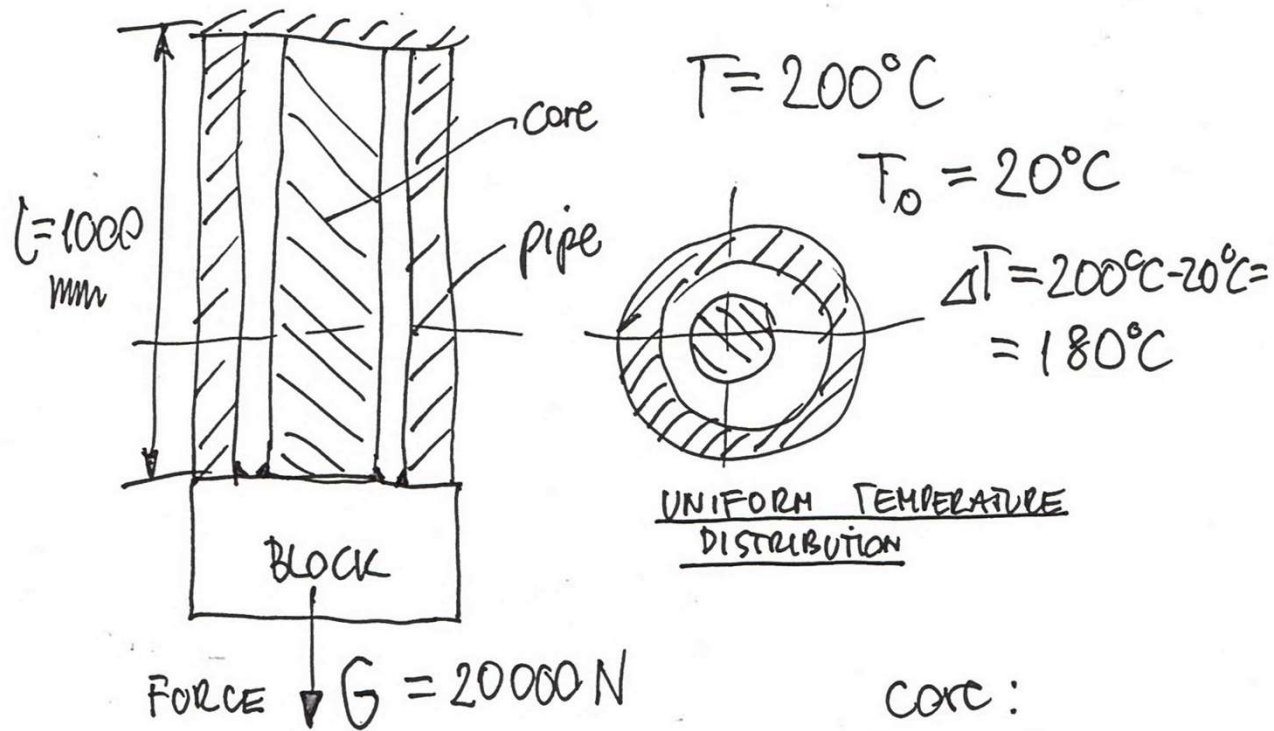
Institute of Aeronautics and Applied Mechanics

Finite element method 2 (FEM 2)

Structural analysis with thermal effect – bar example

10.2021

EXAMPLE : BUILD A FE MODEL OF A STATICALLY INDETERMINATE BAR STRUCTURE. FIND THERMAL LOAD, STRAINS, STRESSES AND REACTIONS IN THE CORE AND PIPE.



pipe:
 (steel)

$$E_p = 2 \cdot 10^5 \text{ MPa}$$

$$\alpha_p = 1.2 \cdot 10^{-5} \frac{1}{^\circ\text{C}}$$

$$A_p = 200 \text{ mm}^2$$

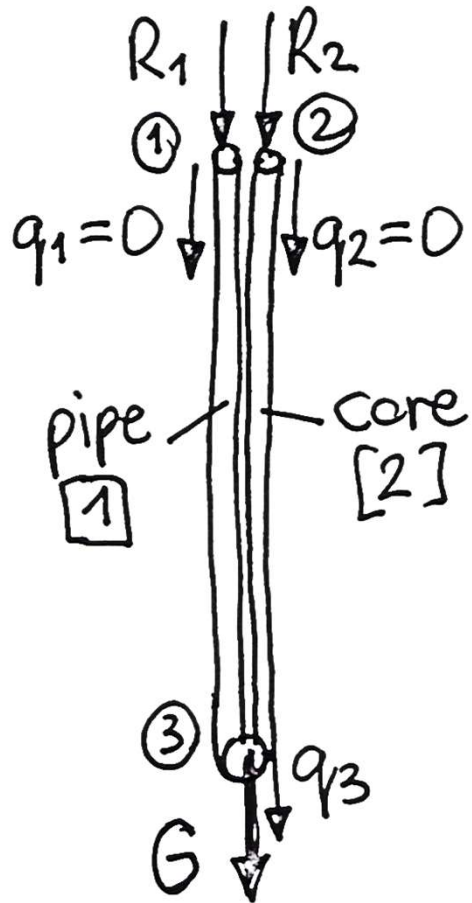
core:
 (copper)

$$E_c = 1,06 \cdot 10^5 \text{ MPa}$$

$$\alpha_c = 1.58 \cdot 10^{-5} \frac{1}{^\circ\text{C}}$$

$$A_c = 50 \text{ mm}^2$$

FE MODEL



$$\{q\} = \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \end{Bmatrix} \quad (\text{mm})$$

3×1

$$\{F_s\} = \begin{Bmatrix} R_1 \\ R_2 \\ G \end{Bmatrix}$$

THERMAL LOAD

① $\Delta T_1 = \Delta T$
 ③ $\Delta T_3 = \Delta T$

$$\{F_T\}_1 = \frac{\Delta T_1 + \Delta T_3}{2} \alpha_p E_p A_p \begin{Bmatrix} -1 \\ 1 \end{Bmatrix} = \Delta T \alpha_p E_p A_p \begin{Bmatrix} -1 \\ 1 \end{Bmatrix}$$

2x1

② $\Delta T_2 = \Delta T$
 ③ $\Delta T_3 = \Delta T$

$$\{F_T\}_2 = \frac{\Delta T_2 + \Delta T_3}{2} \alpha_c E_c A_c \begin{Bmatrix} -1 \\ 1 \end{Bmatrix} = \Delta T \alpha_c E_c A_c \begin{Bmatrix} -1 \\ 1 \end{Bmatrix}$$

③

$$\{F_T\}_1^* = \Delta T \alpha_p E_p A_p \begin{Bmatrix} -1 \\ 0 \\ 1 \end{Bmatrix}$$

3x1

$$\{F_T\}_2^* = \Delta T \alpha_c E_c A_c \begin{Bmatrix} 0 \\ -1 \\ 1 \end{Bmatrix}$$

3x1

$$\begin{Bmatrix} F_T \end{Bmatrix}_{3 \times 1} = \begin{Bmatrix} F_T \end{Bmatrix}_1^* + \begin{Bmatrix} F_T \end{Bmatrix}_2^* = \Delta T \cdot \begin{Bmatrix} -\alpha_p E_p A_p \\ -\alpha_c E_c A_c \\ \alpha_c E_c A_c + \alpha_p E_p A_p \end{Bmatrix}$$

$$[K]_e = \frac{EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$[k]_1 = \frac{E_p A_p}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}, \quad [k]_2 = \frac{E_c A_c}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$[K] = \frac{1}{L} \cdot \begin{bmatrix} E_p A_p & 0 & -E_p A_p \\ 0 & E_c A_c & -E_c A_c \\ -E_p A_p & -E_c A_c & E_p A_p + E_c A_c \end{bmatrix}$$

Boundary conditions:

$$q_1 = 0, q_2 = 0$$

$$\underset{3 \times 3}{[K]} \cdot \underset{3 \times 1}{\{q\}} = \{F_s\} + \{F_T\}$$

$$\left(\frac{E_p A_p + E_c A_c}{L} \right) \cdot q_3 = G + \Delta T \cdot (\alpha_c E_c A_c + \alpha_p E_p A_p)$$

$$q_3 = \frac{G + \Delta T (\alpha_c E_c A_c + \alpha_p E_p A_p)}{\frac{E_p A_p + E_c A_c}{L}} = 0.2682 \text{ mm}$$

TOTAL STRAIN :

$$\text{1st element : } \epsilon_1 = \frac{q_3 - q_1}{l_1} = \frac{q_3}{L} = 2.682 \cdot 10^{-3}$$

$$\text{2nd element : } \epsilon_2 = \frac{q_3 - q_2}{l_2} = \frac{q_3}{L} \Rightarrow \epsilon_1 = \epsilon_2 = \epsilon$$

THERMAL STRAIN:

$$\boxed{1} \quad \epsilon_{T1} = \alpha_p \cdot \Delta T = 2.16 \cdot 10^{-3}$$

$$\boxed{2} \quad \epsilon_{T2} = \alpha_c \cdot \Delta T = 2.844 \cdot 10^{-3}$$

ELASTIC STRAIN:

$$\boxed{1} \quad \epsilon_{e1} = \epsilon - \epsilon_{T1} = 0.522 \cdot 10^{-3} \quad (\text{POSITIVE})$$

$$\boxed{2} \quad \epsilon_{e2} = \epsilon - \epsilon_{T2} = -0.1625 \cdot 10^{-3} \quad (\text{NEGATIVE})$$

STRESS :

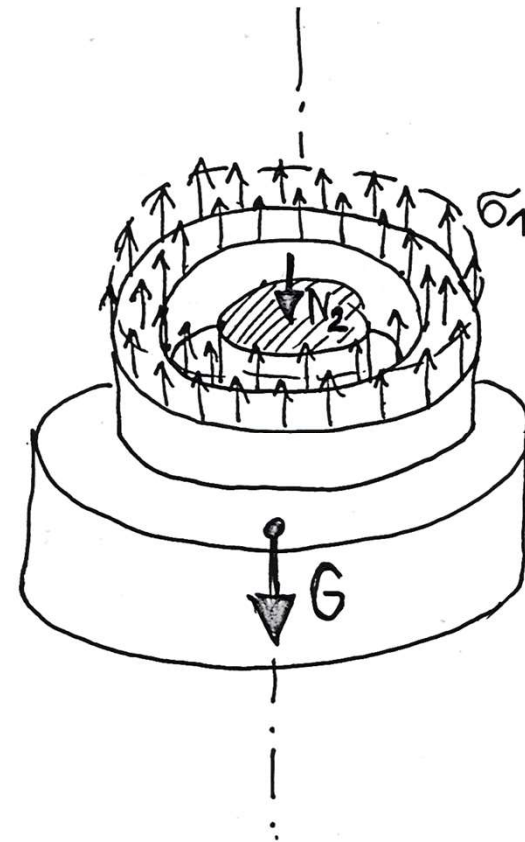
$$\text{1) } \sigma_1 = E_p \cdot \epsilon_{e1} = 104.31 \text{ MPa}$$

$$\text{2) } \sigma_2 = E_c \cdot \epsilon_{e2} = -17.22 \text{ MPa}$$

INTERNAL FORCES

$$\text{1) } N_1 = \sigma_1 \cdot A_p = 20861.1 \text{ N}$$

$$\text{2) } N_2 = \sigma_2 \cdot A_c = -861.1 \text{ N}$$



REACTIONS

$$\begin{matrix} [K] & \cdot & \{q\} & = & \{F\} \\ 3 \times 3 & & 3 \times 1 & & 3 \times 1 \end{matrix}$$

$$\text{I)} \quad \frac{E_p A_p}{L} \cdot q_1 + 0 \cdot q_2 + \left(-\frac{E_p \cdot A_p}{L}\right) q_3 = R_1 - \alpha_p \cdot E_p A_p \Delta T$$

$$\text{II)} \quad 0 \cdot q_1 + \frac{E_c A_c}{L} \cdot q_2 - \frac{E_c A_c}{L} q_3 = R_2 - \alpha_c \cdot E_c A_c \Delta T$$

from I):

$$R_1 = \alpha_p E_p A_p \Delta T - \frac{E_p A_p}{L} \cdot q_3 = -20861.1 \text{ N}$$

from II):

$$R_2 = \alpha_c E_c A_c \Delta T - \frac{E_c A_c}{L} q_3 = 861.1 \text{ N}$$